Symmetry breaking of rigid or flexible splitter plate in a cylinder wake

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Introduction

Experimental results

Theoretical model

Quasi-steady simulation

Conclusion

Spontaneous deviations in fluid/solid systems

Low-Re flow, freely rotating plate



Low-Re flow, freely rotating plate (numerical study)







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ONERA

Spontaneous deviations in fluid/solid systems

Higher-Re flow, freely rotating plate



Low-Re flow, freely rotating plate (numerical study)



Clamped flexible plate (numerical study)



Experimental setup - Soap film channel and visualisation



Test section [1]

- Dimension : 90 cm * 9 cm
- Flowing fluid film (< $10\mu m$)
- *U* = 2.2*m*/*s*

Visualisation - Lightning

- Thin-film interference [2]
- Monochromatic light : green LED 535nm
- White polychromatic light

Visualisation - Acquisition

• Camera : Phantom 7.3 : $F_a = 4000 Hz$



Rutgers *et al.*, RS Inst., 72 (7) (2001)
 Gharib and Beizaie of Visualization 2, 119-126 (1999)

Experimental setup

- Cylinder : D = 3 mm, $Re_D = 350$
- Filament : L/D = [0 9], E = 2 GPa







Quasi-steady simulation

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L/D = 0



L/D = 0.35



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L/D = 0.35



L/D = 1.40



L/D = 6.89

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Experimental results - Static instability





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Simplified theoretical model



Fluid forces [1]

• Fluid force inside the back flow

$$F_n^+ = \mathcal{A}B\left(-U_{bf} + rac{B}{2}\dot{\phi}sin(\phi)
ight)^2sin(\phi)$$

• Fluid force outside the back flow

$$F_n^- = -\mathcal{A}(L-B)\left(U_f + \frac{L+B}{2}\dot{\phi}sin(\phi)\right)^2sin(\phi)$$



Quasi-steady simulation

Theoretical model



Filament drag force \implies Moment opposed to filament motion

$$F_d = \pm \mathcal{A} L \left(\frac{L}{2} \dot{\phi}\right)^2$$

Stiffness restoring force \implies Stabilising moment $F_r = -\frac{1}{2}KL\sin(\phi)$





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Moment equation

At the filament hinge point, projected along the filament axis :

$$\ddot{\phi} = \frac{1}{l_0} \left(F_n^+ \left[\frac{B}{2} \right] + F_n^- \left[\frac{B}{2} + \frac{L}{2} \right] + F_r \left[\frac{2L}{3} \right] + F_d \left[\frac{2L}{3} \right] \right)$$



Quasi-steady simulation

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Theoretical model - Results : static instability





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Quasi-steady simulations

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Derivation of the mode

Dynamics of the solid is described with only one time-dependent variable lpha(t),

Solid displacement field :

$$ec{\xi}(ec{x},t)\simeq lpha(t)ec{\phi}(ec{x})$$

with $\vec{\phi}(\vec{x})$: first bending mode of the solid.

Projection of the fluid-solid dynamics on this vibration mode

$$M\frac{\mathrm{d}^{2}\alpha}{\mathrm{d}t^{2}} + K(E_{s})\alpha = F(t,\alpha) \tag{1}$$

with M: the modal mass coefficient, K: modal stiffness coefficient, F: modal projection of the aerodynamic forces



Quasi-steady simulations

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Derivation of the mode

Hypotheses : 2 time scales + Taylor decomposition

$$\frac{\mathrm{d} < F >}{\mathrm{d}\alpha} \bigg|_{0} \simeq \underbrace{\left\{ \frac{1}{\varepsilon} \int_{\Gamma(\varepsilon)} < \boldsymbol{\sigma}(\boldsymbol{u}, \boldsymbol{p}) \boldsymbol{n} > \cdot \boldsymbol{\phi} \, \mathrm{d}\Gamma \right\}}_{\delta F} \alpha$$

with $< \sigma(u, p) n >$: mean aerodynamic load

$$M\frac{\mathrm{d}^{2}\alpha}{\mathrm{d}t^{2}} + (K(E_{s}) - \delta F)\alpha = 0.$$
⁽²⁾

- quasi-static deviation appears as a divergence instability
- when the threshold $K(E_s) = \delta F$ is crossed
- i.e. when the <u>added stiffness</u> force generated through the interaction with the mean flow balances the restoring elastic force



Quasi-steady simulations

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Numerical methods and results

Methodology

- Time-average loads exerted by the fluid on the rigid, deformed configurations extracted from time-marching simulations
- Incompressible NS equations solved using a semi-implicit solver





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Experimental study

• Enlighten a bifurcation at low filament aspect ratio







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Phenomelogical model

- Adding stiffness restoring force
- Captures the bifurcation at low L/D
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Quasi-steady model

- The 2 bifurcations are captured
- At Re=250, thresholds not the same as in the experiments
- Simulations at experimental Re=350 are ongoing ...





Thank you for you attention





